SECTION 2: BLOCK DIAGRAMS & SIGNAL FLOW GRAPHS

ESE 430 – Feedback Control Systems



Block Diagrams

 In the introductory section we saw examples of *block diagrams* to represent systems, e.g.:



- Block diagrams consist of
 - Blocks these represent subsystems typically modeled by, and labeled with, a transfer function
 - Signals inputs and outputs of blocks signal direction indicated by arrows – could be voltage, velocity, force, etc.
 - Summing junctions points were signals are algebraically summed subtraction indicated by a negative sign near where the signal joins the summing junction

Standard Block Diagram Forms

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The basic input/output relationship for a single block is:



 $Y(s) = U(s) \cdot G(s)$

- Block diagram blocks can be connected in three basic forms:
 - Cascade
 - Parallel
 - Feedback
- We'll next look at each of these forms and derive a singleblock equivalent for each

Cascade Form

Blocks connected in *cascade*:



$$G_{eq}(s) = G_1(s) \cdot G_2(s) \cdot G_3(s)$$

The equivalent transfer function of cascaded blocks is the product of the individual transfer functions

Parallel Form

Blocks connected in parallel:



 $X_1(s) = U(s) \cdot G_1(s)$ $X_2(s) = U(s) \cdot G_2(s)$ $X_3(s) = U(s) \cdot G_3(s)$ $Y(s) = X_1(s) \pm X_2(s) \pm X_3(s)$

$$Y(s) = U(s) \cdot G_1(s) \pm U(s) \cdot G_2(s) \pm U(s) \cdot G_3(s)$$
$$Y(s) = U(s)[G_1(s) \pm G_2(s) \pm G_3(s)] = U(s) \cdot G_{eq}(s)$$
$$G_{eq}(s) = G_1(s) \pm G_2(s) \pm G_3(s)$$

The equivalent transfer function is the sum of the individual transfer functions:



Feedback Form

□ Of obvious interest to us, is the *feedback form*:



$$Y(s) = E(s)G(s)$$

$$Y(s) = [R(s) - X(s)]G(s)$$

$$Y(s) = [R(s) - Y(s)H(s)]G(s)$$

$$Y(s)[1 + G(s)H(s)] = R(s)G(s)$$
$$Y(s) = R(s) \cdot \frac{G(s)}{1 + G(s)H(s)}$$

 \Box The *closed-loop transfer function*, T(s), is

$$T(s) = \frac{Y(s)}{R(s)} = \frac{G(s)}{1 + G(s)H(s)}$$

Feedback Form



Note that this is *negative feedback*, for *positive feedback*:

$$T(s) = \frac{G(s)}{1 - G(s)H(s)}$$

- □ The G(s)H(s) factor in the denominator is the **loop gain** or **open-loop transfer function**
- □ The gain from input to output with the feedback path broken is the *forward path gain* here, G(s)

□ In general:

$$T(s) = \frac{\text{forward path gain}}{1 - \text{loop gain}}$$

Closed-Loop Transfer Function - Example

Calculate the closed-loop transfer function



- \square D(s) and G(s) are in cascade
- □ $H_1(s)$ is in cascade with the feedback system consisting of D(s), G(s), and $H_2(s)$

$$T(s) = H_1(s) \cdot \frac{D(s)G(s)}{1 + D(s)G(s)H_2(s)}$$
$$T(s) = \frac{H_1(s)D(s)G(s)}{1 + D(s)G(s)H_2(s)}$$

Unity-Feedback Systems

We're often interested in unity-feedback systems



Feedback path gain is unity

Can always reconfigure a system to unity-feedback form

Closed-loop transfer function is:

$$T(s) = \frac{D(s)G(s)}{1 + D(s)G(s)}$$

¹¹ Block Diagram Manipulation

Block Diagram Algebra

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- Often want to simplify block diagrams into simpler, recognizable forms
 - **•** To determine the equivalent transfer function
- Simplify to instances of the three standard forms, then simplify those forms
- Move blocks around relative to summing junctions
 and pickoff points simplify to a standard form
 - Move blocks forward/backward past summing junctions
 - Move blocks forward/backward past pickoff points

Moving Blocks Back Past a Summing Junction

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The following two block diagrams are equivalent:



 $Y(s) = [U_1(s) + U_2(s)]G(s) = U_1(s)G(s) + U_2(s)G(s)$

Moving Blocks Forward Past a Summing Junction

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The following two block diagrams are equivalent:



$$Y(s) = U_1(s)G(s) + U_2(s) = \left[U_1(s) + U_2(s)\frac{1}{G(s)}\right]G(s)$$

Moving Blocks Relative to Pickoff Points

We can move blocks backward past pickoff points:



And, we can move them forward past pickoff points:



Rearrange the following into a unity-feedback system



- □ Move the feedback block, H(s), forward, past the summing junction
- Add an inverse block on R(s) to compensate for the move



Closed-loop transfer function:

$$T(s) = \frac{\frac{1}{H(s)}H(s)G(s)}{1 + G(s)H(s)} = \frac{G(s)}{1 + G(s)H(s)}$$

 Find the closed-loop transfer function of the following system through block-diagram simplification



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 \Box $G_1(s)$ and $H_1(s)$ are in feedback form



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Move $G_2(s)$ backward past the pickoff point



□ Block from previous step, $G_2(s)$, and $H_2(s)$ become a feedback system that can be simplified

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- Simplify the feedback subsystem
- □ Note that we've dropped the function of s notation, (s), for clarity



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Simplify the two parallel subsystems



$$G_{eq}(s) = G_3 + \frac{G_4}{G_2}$$

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- Now left with two cascaded subsystems
 - Transfer functions multiply

$$G_{eq}(s) = \frac{G_1 G_2 G_3 + G_1 G_4}{1 - G_1 H_1 + G_1 G_2 H_2}$$



The equivalent, close-loop transfer function is

$$T(s) = \frac{G_1 G_2 G_3 + G_1 G_4}{1 - G_1 H_1 + G_1 G_2 H_2}$$

²⁴ Multiple-Input Systems

Multiple Input Systems

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- Systems often have more than one input
 E.g., reference, R(s), and disturbance, W(s)



Two transfer functions:From reference to output

$$T(s) = Y(s)/R(s)$$

From disturbance to output

$$T_w(s) = Y(s)/W(s)$$

Transfer Function – Reference

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- Find transfer function from R(s) to Y(s)
 A linear system superposition applies
 Set W(s) = 0



$$T(s) = \frac{Y(s)}{R(s)} = \frac{D(s)G(s)}{1 + D(s)G(s)}$$

Transfer Function – Reference

Next, find transfer function from W(s) to Y(s)
 Set R(s) = 0

System now becomes:



$$T_{w}(s) = \frac{Y(s)}{W(s)} = \frac{G_{w}(s)G(s)}{1 + D(s)G(s)}$$

Multiple Input Systems

Two inputs, two transfer functions

$$T(s) = \frac{D(s)G(s)}{1+D(s)G(s)}$$
 and $T_w(s) = \frac{G_w(s)G(s)}{1+D(s)G(s)}$

- D(s) is the controller transfer function
 Ultimately, we'll determine this
 We have control over both T(s) and T_w(s)
- What do we want these to be?
 Design T(s) for desired performance
 Design T_w(s) for disturbance rejection



Signal Flow Graphs

An alternative to block diagrams for graphically describing systems



- □ Signal flow graphs consist of:
 - Nodes represent signals
 - Branches represent system blocks
- Branches labeled with system transfer functions
- Nodes (sometimes) labeled with signal names
- Arrows indicate signal flow direction
- Implicit summation at nodes
 - Always a positive sum
 - Negative signs associated with branch transfer functions

Block Diagram → Signal Flow Graph

- To convert from a block diagram to a signal flow graph:
 - 1. Identify and label all signals on the block diagram
 - 2. Place a node for each signal
 - 3. Connect nodes with branches in place of the blocks
 - Maintain correct direction
 - Label branches with corresponding transfer functions
 - Negate transfer functions as necessary to provide negative feedback
 - 4. If desired, simplify where possible

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Convert to a signal flow graph



Label any unlabeled signals
 Place a node for each signal





Connect nodes with branches, each representing a system block



□ Note the -1 to provide negative feedback of $X_2(s)$



- Nodes with a single input and single output can be eliminated, if desired
 - **This makes sense for** $X_1(s)$ and $X_2(s)$
 - Leave U(s) to indicate separation between controller and plant



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- Revisit the block diagram from earlier

• Convert to a signal flow graph



 \Box Label all signals, then place a node for each $x_{s(s)}$





Connect nodes with branches





□ Simplify – eliminate $X_5(s)$, $X_6(s)$, $X_7(s)$, and $X_8(s)$



Signal Flow Graphs vs. Block Diagrams

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- Signal flow graphs and block diagrams are alternative, though equivalent, tools for graphical representation of interconnected systems
- □ A generalization (not a rule)
 - Signal flow graphs more often used when dealing with state-space system models
 - Block diagrams more often used when dealing with transfer function system models

³⁹ Mason's Rule

Mason's Rule

- We've seen how to reduce a complicated block diagram to a single input-to-output transfer function
 - Many successive simplifications
- Mason's rule provides a formula to calculate the same overall transfer function
 - Single application of the formula
 - Can get complicated
- Before presenting the Mason's rule formula, we need to define some terminology

Loop Gain





- Loop gain total gain (product of individual gains) around any path in the signal flow graph
 - Beginning and ending at the same node
 - Not passing through any node more than once
- Here, there are three loops with the following gains:
 - 1. $-G_1H_3$
 - 2. G_2H_1
 - $3. \quad -G_2G_3H_2$

Forward Path Gain



- Forward path gain gain along any path from the input to the output
 - Not passing through any node more than once
- Here, there are two forward paths with the following gains:
 - 1. $G_1 G_2 G_3 G_4$
 - 2. $G_1G_2G_5$

Non-Touching Loops



Non-touching loops – loops that do not have any nodes in common

🗆 Here,

- 1. $-G_1H_3$ does not touch G_2H_1
- 2. $-G_1H_3$ does not touch $-G_2G_3H_2$

Non-Touching Loop Gains



- Non-touching loop gains the product of loop gains from non-touching loops, taken two, three, four, or more at a time
- Here, there are only two pairs of non-touching loops

$$1. \quad [-G_1H_3] \cdot [G_2H_1]$$

2. $[-G_1H_3] \cdot [-G_2G_3H_2]$

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Mason's Rule

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$$T(s) = \frac{Y(s)}{R(s)} = \frac{1}{\Delta} \sum_{k=1}^{P} T_k \Delta_k$$

where

- P = # of forward paths
- $T_k =$ gain of the k^{th} forward path
- $\Delta = 1 \Sigma(\text{loop gains})$

+ Σ (non-touching loop gains taken two-at-a-time)

 $-\Sigma$ (non-touching loop gains taken three-at-a-time)

+ Σ (non-touching loop gains taken four-at-a-time)

 $-\Sigma$...

 $\Delta_k = \Delta - \Sigma$ (loop gain terms in Δ that touch the k^{th} forward path)

Mason's Rule - Example



P = 2

- Forward path gains:
 - $T_1 = G_1 G_2 G_3 G_4$ $T_2 = G_1 G_2 G_5$
- \Box Σ (loop gains):
 - $-G_1H_3 + G_2H_1 G_2G_3H_2$

 $\Sigma(\text{NTLGs taken two-at-a-time}):$ $(-G_1H_3G_2H_1) + (G_1H_3G_2G_3H_2)$

Δ:

 $\Delta = 1 - (-G_1H_3 + G_2H_1 - G_2G_3H_2)$ $+ (-G_1H_3G_2H_1 + G_1H_3G_2G_3H_2)$

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Mason's Rule – Example - Δ_k

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- Simplest way to find Δ_k terms is to calculate Δ with the kth path removed must remove *nodes* as well



With forward path 1 removed, there are no loops, so

$$\Delta_1 = 1 - 0$$
$$\Delta_1 = 1$$

Mason's Rule – Example - Δ_k

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Similarly, removing forward path 2 leaves no loops, so

$$\begin{array}{l} \Delta_2 = 1 - 0\\ \Delta_2 = 1 \end{array}$$

Mason's Rule - Example

For our example:

 $\Delta_1 =$

$$P = 2$$

$$T_1 = G_1 G_2 G_3 G_4$$

$$T_2 = G_1 G_2 G_5$$

$$\Delta = 1 + G_1 H_3 - G_2 H_1 + G_2 G_3 H_2 - G_1 H_3 G_2 H_1 + G_1 H_3 G_2 G_3 H_2$$

$$\Delta_1 = 1$$

$$\Delta_2 = 1$$

$$T(s) = \frac{T_1 \Delta_1 + T_2 \Delta_2}{\Delta}$$
$$T(s) = \frac{G_1 G_2 G_3 G_4 + G_1 G_2 G_5}{1 + G_1 H_3 - G_2 H_1 + G_2 G_3 H_2 - G_1 H_3 G_2 H_1 + G_1 H_3 G_2 G_3 H_2}$$

D



Controller Design – Preview

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- We now have the tools necessary to determine the transfer function of closed-loop feedback systems
- Let's take a closer look at how feedback can help us achieve a desired response

■ Just a preview – this is the objective of the whole course

Consider a simple first-order system

$$\begin{array}{c|c} U(s) & \underline{Plant} & Y(s) \\ \hline & \underline{1} & \\ \hline & s+2 \end{array}$$

□ A single real pole at $s = -2 \frac{rad}{sec}$

Open-Loop Step Response

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This system

 exhibits the
 expected first order step
 response
 No overshoot or
 ringing



Add Feedback

Now let's enclose the system in a feedback loop



Add controller block with transfer function D(s)
 Closed-loop transfer function becomes:

$$T(s) = \frac{D(s)\frac{1}{s+2}}{1+D(s)\frac{1}{s+2}} = \frac{D(s)}{s+2+D(s)}$$

Clearly the addition of feedback and the controller changes the transfer function – but how?
 Let's consider a couple of example cases for D(s)

Add Feedback

First, consider a simple gain block for the controller



- Error signal, E(s), amplified by a constant gain, K_C
 A proportional controller, with gain K_C
- Now, the closed-loop transfer function is:

$$T(s) = \frac{\frac{K_C}{s+2}}{1 + \frac{K_C}{s+2}} = \frac{K_C}{s+2 + K_C}$$

- □ A single real pole at $s = -(2 + K_C)$
 - Pole moved to a higher frequency
 - A faster response

Open-Loop Step Response



- As feedback gain increases:
 - Pole moves to a higher frequency
 - Response gets faster



First-Order Controller

Next, allow the controller to have some dynamics of its own



- □ Now the controller is a first-order block with gain K_C and a pole at s = -b
- □ This yields the following closed-loop transfer function:

$$T(s) = \frac{\frac{K_C}{(s+b)}\frac{1}{(s+2)}}{1 + \frac{K_C}{(s+b)}\frac{1}{(s+2)}} = \frac{K_C}{s^2 + (2+b)s + 2b + K_C}$$

- □ The closed-loop system is now *second-order*
 - One pole from the plant
 - One pole from the controller

First-Order Controller



$$T(s) = \frac{K_C}{s^2 + (2+b)s + 2b + K_C}$$

Two closed-loop poles:

$$s_{1,2} = -\frac{(b+2)}{2} \pm \frac{\sqrt{b^2 - 4b + 4 - 4K_C}}{2}$$

- \Box Pole locations determined by *b* and *K*_{*C*}
 - Controller parameters we have control over these
 - Design the controller to place the poles where we want them
- □ So, where do we want them?
 - Design to performance specifications
 - Risetime, overshoot, settling time, etc.

Design to Specifications

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The second-order closed-loop transfer function

$$T(s) = \frac{K_C}{s^2 + (2+b)s + 2b + K_C}$$

can be expressed as

$$T(s) = \frac{K_C}{s^2 + 2\zeta\omega_n s + \omega_n^2} = \frac{K_C}{s^2 + 2\sigma s + \omega_n^2}$$

- Let's say we want a closed-loop response that satisfies the following specifications:
 - $\bullet \% OS \le 5\%$
 - $\bullet t_s \le 600 \, msec$
- Use %OS and t_s specs to determine values of ζ and σ
 Then use ζ and σ to determine K_c and b

Determine ζ from Specifications

Overshoot and damping ratio, ζ, are related as follows:

$$\zeta = \frac{-\ln(OS)}{\sqrt{\pi^2 + \ln^2(OS)}}$$

□ The requirement is $\% OS \le 5\%$, so

$$\zeta \ge \frac{-\ln(0.05)}{\sqrt{\pi^2 + \ln^2(0.05)}} = 0.69$$

□ Allowing some margin, set $\zeta = 0.75$

Determine σ from Specifications

Settling time (±1%) can be approximated from σ as

$$t_s \approx \frac{4.6}{\sigma}$$

- □ The requirement is $t_s \leq 600 \text{ msec}$
- □ Allowing for some margin, design for $t_s = 500 \text{ msec}$

$$t_s \approx \frac{4.6}{\sigma} = 500 \ msec \rightarrow \sigma = \frac{4.6}{500 \ msec}$$

which gives

$$\sigma = 9.2 \frac{rad}{sec}$$

 $\hfill\square$ We can then calculate the natural frequency from ζ and σ

$$\omega_n = \frac{\sigma}{\zeta} = \frac{9.2}{0.75} = 12.27 \frac{rad}{sec}$$

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□ The characteristic polynomial is

$$s^{2} + (2+b)s + 2b + K_{c} = s^{2} + 2\sigma s + \omega_{n}^{2}$$

□ Equating coefficients to solve for *b*:

$$2 + b = 2\sigma = 18.4$$

 $b = 16.4$

and *K_c*:

$$2b + K_c = \omega_n^2 = (12.27)^2 = 150.5$$

 $K_c = 150.5 - 2 \cdot 16.4 = 117.7 \rightarrow 118$
 $K_c = 118$

□ The controller transfer function is

$$D(s) = \frac{118}{s + 16.4}$$

Closed-Loop Poles

- Closed-loop system
 is now second order
- Controller designed to place the two closed-loop poles at desirable locations:

•
$$s_{1,2} = -9.2 \pm j8.13$$

• $\zeta = 0.75$
• $\omega_n = 12.3$



Closed-Loop Step Response

- Closed-loop step
 response satisfies
 the specifications
- Approximations
 were used
 - If requirements not met *iterate*



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