

SECTION 3: PUMPED-HYDRO ENERGY STORAGE

2

Introduction

Potential Energy Storage

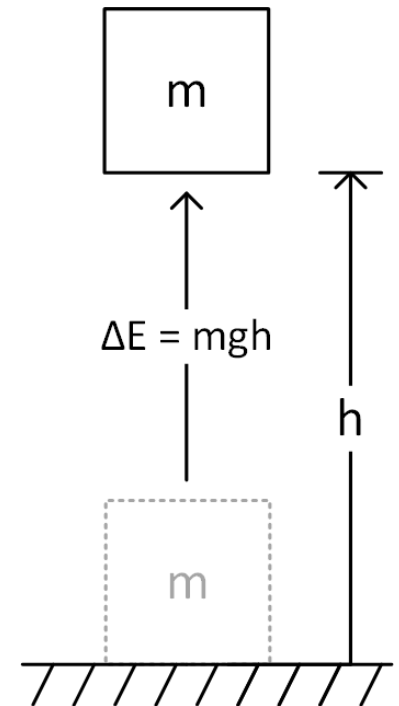
3

- Energy can be stored as ***potential energy***
- Consider a mass, m , elevated to a height, h
- Its potential energy increase is

$$E = mgh$$

where $g = 9.81 \text{ m/s}^2$ is gravitational acceleration

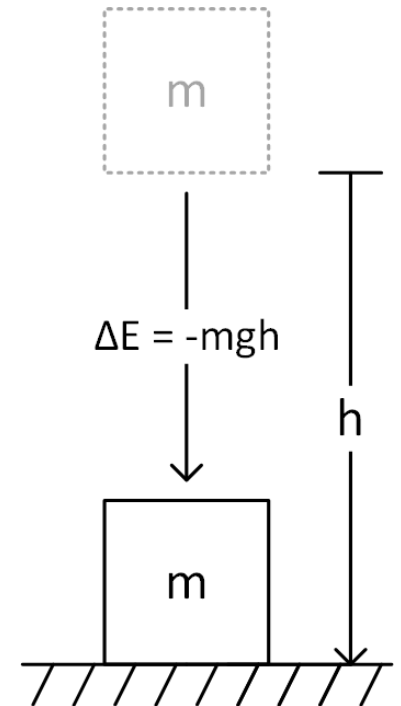
- Lifting the mass requires an input of work equal to (at least) the energy increase of the mass
 - We put energy in to lift the mass
 - That energy is stored in the mass as potential energy



Potential Energy Storage

4

- If we allow the mass to fall back to its original height, we can capture the stored potential energy
 - Potential energy converted to kinetic energy as the mass falls
 - Kinetic energy can be captured to perform work
 - Perhaps converted to rotational energy, and then to electrical energy

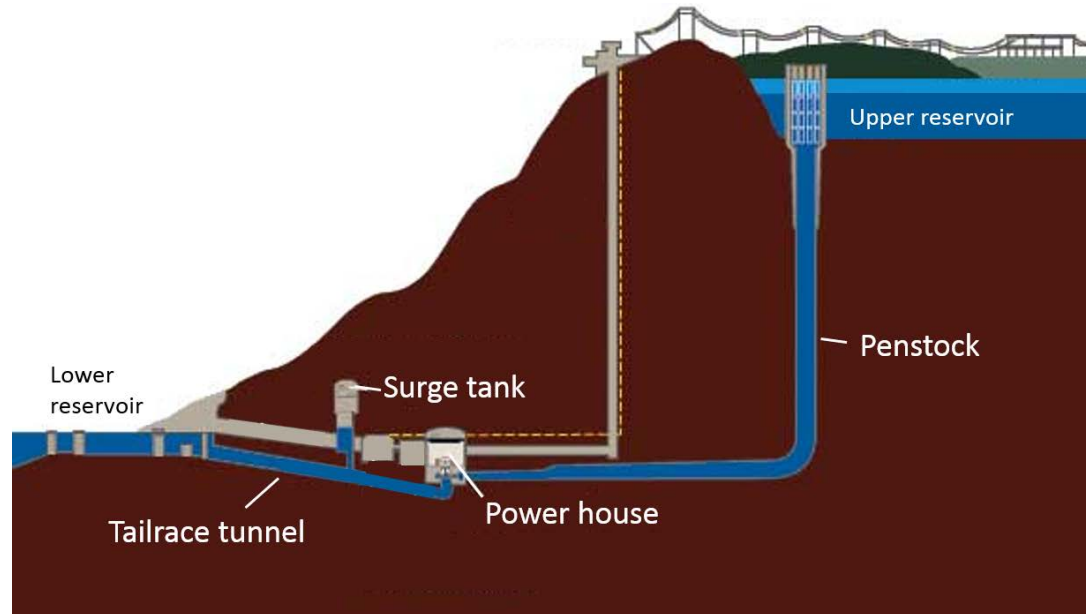


Pumped-Hydro Energy Storage

5

- Potential energy storage in elevated mass is the basis for ***pumped-hydro energy storage (PHES)***

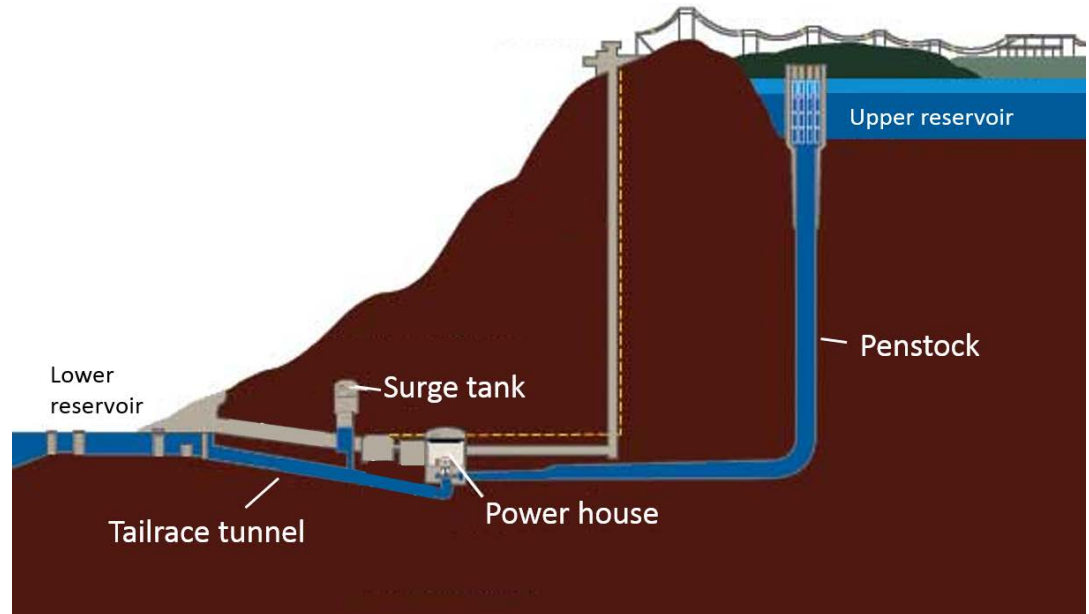
- Energy used to pump water from a lower reservoir to an upper reservoir
- ***Electrical energy*** input to ***motors*** converted to ***rotational mechanical energy***
- ***Pumps*** transfer energy to the water as ***kinetic***, then ***potential energy***



Pumped-Hydro Energy Storage

6

- Energy stored in the water of the upper reservoir is released as water flows to the lower reservoir
 - ▣ Potential energy converted to kinetic energy
 - ▣ Kinetic energy of falling water turns a turbine
 - ▣ Turbine turns a generator
 - ▣ Generator converts mechanical energy to electrical energy



History of PHES

7

- PHES first introduced in Italy and Switzerland in the 1890's
 - Favorable topography in the Alps
 - Four-unit (quaternary) systems
 - Turbine
 - Generator
 - Motor
 - Pump

History of PHES

8

- First PHES plant in the US:
 - Rocky River hydro plant, New Milford, CT
 - Water from the Housatonic River pumped up into Candlewood Lake
 - 230 feet of head
 - 6 billion ft^3 of water
 - Two-unit (binary) system
 - Reversible pump/turbine – one of the first
 - 29 MW of generating power



Pumped-Hydro Storage Today

9

- PHES accounts for 99% of worldwide energy storage
 - Total power: ~127 GW
 - Total energy: ~740 TWh
 - Power of individual plants: 10s of MW – 3 GW
- In the US:
 - ~40 operational PHES plants
 - 75% are > 500 MW – strong economies of scale
 - Total power: ~23 GW
 - Current plans for an additional ~6 GW
 - Total energy: ~220 TWh

10

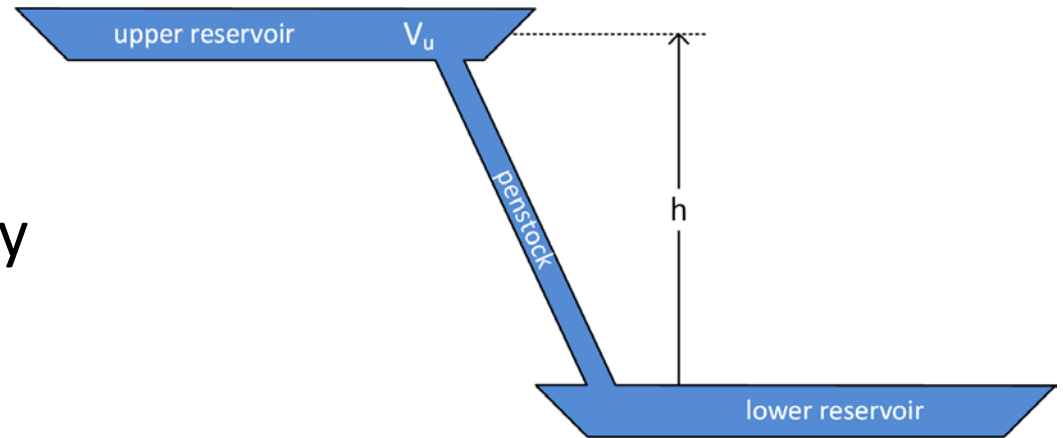
PHES Fundamentals

PHES Fundamentals

11

- Two storage reservoirs

- Upper and lower
- Lower reservoir may be a river or even the sea



- Separated by a height, h

- The hydraulic head
- Assume $h \gg$ depth of the upper reservoir
 - h remains constant throughout charge/discharge cycle

- Upper reservoir can store a volume of water, V_u

PHES Fundamentals - Energy

12

- **Total stored energy** (assuming it is all at a height, h)

$$E_t = mgh = V_u \rho gh$$

where $\rho = 1000 \text{ kg/m}^3$ is the density of water

- Verifying that we do, in fact, have units of energy

$$[E_t] = m^3 \frac{\text{kg}}{m^3} \frac{m}{s^2} m = \frac{\text{kg} \cdot m}{s^2} m = N \cdot m = J$$

- The **energy density** – energy per unit volume – of the stored water is therefore

$$e_v = \frac{E_t}{V_u} = \rho gh$$

$$[e_v] = \frac{\text{kg}}{m^3} \frac{m}{s^2} m = \frac{\text{kg} \cdot m^2}{s^2} \frac{1}{m^3} = \frac{J}{m^3}$$

PHES Fundamentals – Hydrostatic Pressure

13

- The energy density of the stored water is also the ***hydrostatic pressure*** at the level of the lower reservoir

$$p = \rho g h$$

$$[p] = \frac{kg}{m^3} \frac{m}{s^2} m = \frac{kg \cdot m}{s^2} \frac{1}{m^2} = \frac{N}{m^2} = Pa$$

- This is the ***energy density*** of the water at the turbine

PHES Fundamentals - Power

14

- The rate at which energy is transferred to the turbine (from the pump) is the power extracted from (delivered to) the water

$$P = e_v Q = pQ = \rho ghQ$$

where Q is the **volumetric flow rate** of the water

$$[P] = \frac{J}{m^3} \frac{m^3}{s} = \frac{J}{s} = W$$

- This is the total power available at the turbine
 - Greater than (less than) the power actually delivered to the turbine (from the pump), due to inefficiencies

A Generalized Power Relation

15

- Note that **power** is given by the product of a driving potential, or **effort**, p , and a **flow**, Q

$$P = pQ$$

- Similar to power for a **translational mechanical** system

$$P = Fv$$

where the effort is force, F , and the flow is velocity, v

- Or, a **rotational mechanical** system

$$P = \tau\omega$$

where the effort is torque, τ , and the flow is angular velocity, ω

A Generalized Power Relation

16

- Also similar to an *electrical* system

$$P = VI$$

where the effort is voltage, V , and the flow is current, I

- In general, for systems in any energy domain, ***power is given by the product of effort and flow***

$$P = e \cdot f$$

Energy & Power vs. Head

17

- The total stored energy and available power are

$$E_t = V_u \rho g h$$

$$P = \rho g h Q$$

- Both are proportional to head, h
 - ▣ Large vertical separation between lower and upper reservoirs is desirable
 - ▣ Limited by topography
 - ▣ Limited by equipment – pump and turbine
- **Specific energy** is also proportional to head:

$$e_m = \frac{E_t}{m_u} = \frac{E_t}{V_u \rho} = \frac{V_u \rho g h}{V_u \rho} = g h$$

- As is **energy density**:

$$e_v = \frac{E_t}{V_u} = \rho g h$$

Specific Energy & Energy Density vs. Head

18

- Most PHES plants have head in the range of 100 – 1000 m
- Using **300 m** as a representative head, gives:
 - ▣ **Energy density for $h = 300$ m:**

$$e_v = \rho gh = 1000 \frac{\text{kg}}{\text{m}^3} \cdot 9.81 \frac{\text{m}}{\text{s}^2} \cdot 300 \text{ m}$$

$$e_v = 2.9 \frac{\text{MJ}}{\text{m}^3} \cdot \frac{1}{3600} \frac{\text{Wh}}{\text{J}} = \mathbf{818 \frac{Wh}{m^3}}$$

$$e_v = 818 \frac{\text{Wh}}{\text{m}^3} \cdot 1 \frac{\text{m}^3}{1000 \text{ L}} = \mathbf{0.818 \frac{Wh}{L}}$$

- ▣ **Specific energy for $h = 300$ m:**

$$e_m = gh = 9.81 \frac{\text{m}}{\text{s}^2} \cdot 300 \text{ m} = 4905 \frac{\text{m}^2}{\text{s}^2} = 2.9 \frac{\text{kJ}}{\text{kg}}$$

$$e_m = 2.9 \frac{\text{kJ}}{\text{kg}} \cdot \frac{1}{3600} \frac{\text{Wh}}{\text{J}} = \mathbf{0.818 \frac{Wh}{kg}}$$

Specific Energy & Energy Density

19

- Comparison of PHES energy density and specific energy with other energy storage/sources

| | PHES h = 100 m | PHES h = 500 m | PHES h = 1000m | Li-ion Battery | Natural Gas | Gasoline | Units |
|----------------------------|-------------------|-------------------|-------------------|-------------------|----------------|----------|-------|
| Energy Density | 0.273 | 1.36 | 2.73 | 400 | 10.1 | 9,500 | Wh/L |
| Specific Energy | 0.273 | 1.36 | 2.73 | 150 | 15,400 | 13,000 | Wh/kg |

- Even at high heads, PHES has very low energy density
 - Large reservoirs are required

PHES Applications

20

- Pumped hydro plants can supply large amounts of both ***power*** and ***energy***
- Can ***quickly respond to large load variations***
- Uses for PHES:

- ***Peak shaving/load leveling***
 - Help meet loads during peak hours
 - Generating while releasing water from upper reservoir
 - Supplying expensive energy
 - Store energy during off-peak hours
 - Pumping water to the upper reservoir
 - Consuming inexpensive energy

PHES Applications

21

□ ***Frequency regulation***

- Power variation to track short-term load variations
- Helps maintain grid frequency at 60 Hz (50 Hz)

□ ***Voltage support***

- Reactive power flow control to help maintain desired grid voltage
 - Varying the field excitation voltage of the generator/motor
- Even at zero real power – not pumping or generating – unloaded motor/generator can serve as synchronous condenser
 - Pump/turbine spinning in air

PHES Applications

22

□ ***Black start capability***

- ▣ Ability to start generating without an external power supply
- ▣ Bring the grid back online after a blackout

□ ***Spinning reserve***

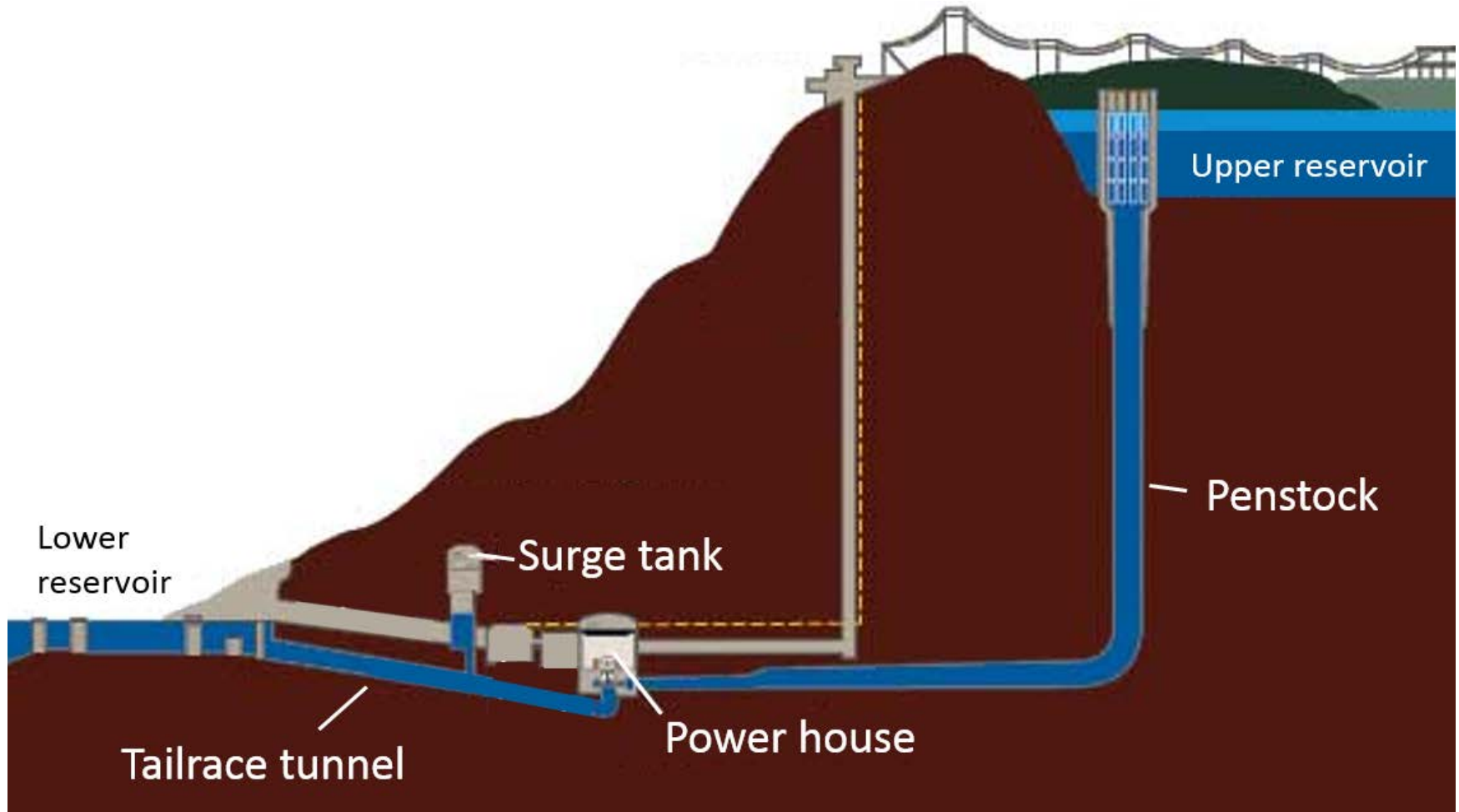
- ▣ Spare online generating capacity
- ▣ Capable of responding quickly – within seconds to minutes – to the need for additional generation

23

Components of a PHES Plant

Components of a PHES Plant

24



PHES Components – Reservoirs

25

- ***Upper and lower reservoirs*** separated by an elevation difference

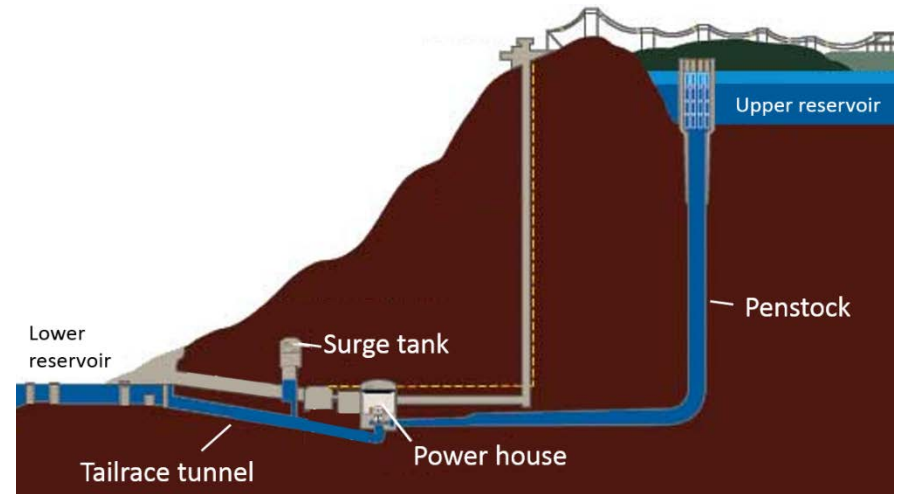
- Two configurations:

- ***Open-loop:***

- At least one of the reservoirs connected to a source of natural inflow
 - Natural lake, river, river-fed reservoir, the sea

- ***Closed-loop:***

- Neither reservoir has a natural source of inflow
 - Initial filling and compensation of leakage and evaporation provided by ground water wells
 - Less common than open-loop

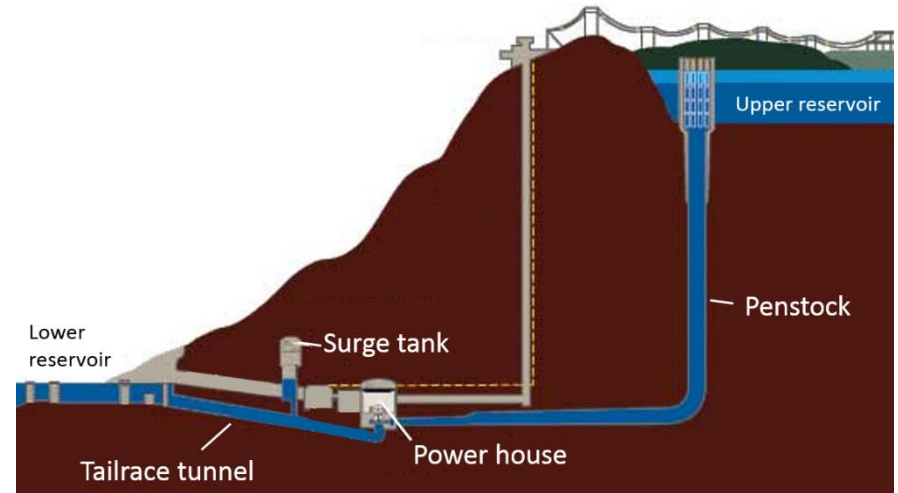


PHES Components – Penstock

26

□ **Penstock**

- Conduit for water flowing between reservoirs and to the pump/generator
- Above-ground pipes or below ground shafts/tunnels
 - 5 -10 m diameter is common
 - One plant may have several penstocks
 - Typically steel- or concrete-lined, though may be unlined
- Flow velocity range of 1 – 5 m/s is common
- Tradeoff between cost and efficiency for a given flow rate, Q
 - Larger cross-sectional area:
 - Slower flow
 - Lower loss
 - Higher cost

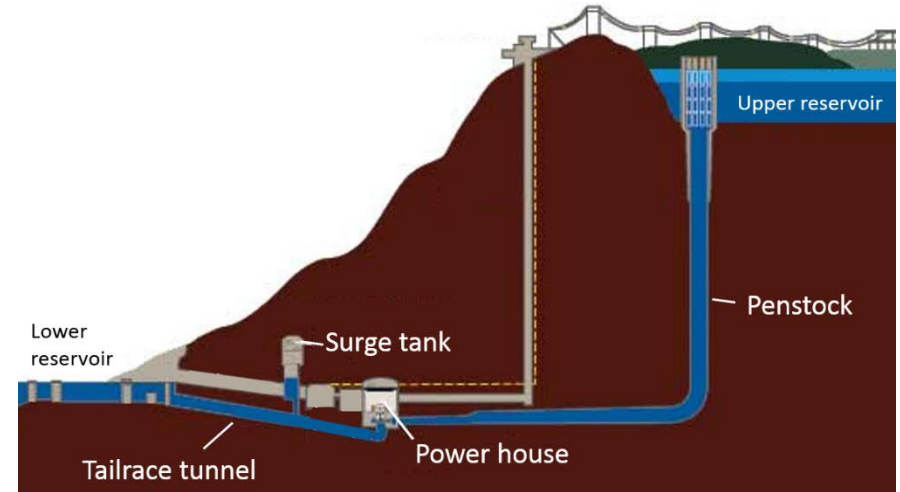


PHES Components

27

□ ***Tailrace tunnel***

- Typically, larger diameter than penstocks
- Lower pressure
- Lower flow rate
- Downward slope from lower reservoir to pump/turbine
 - Inlet head helps prevent cavitation in pumping mode



□ ***Surge tanks***

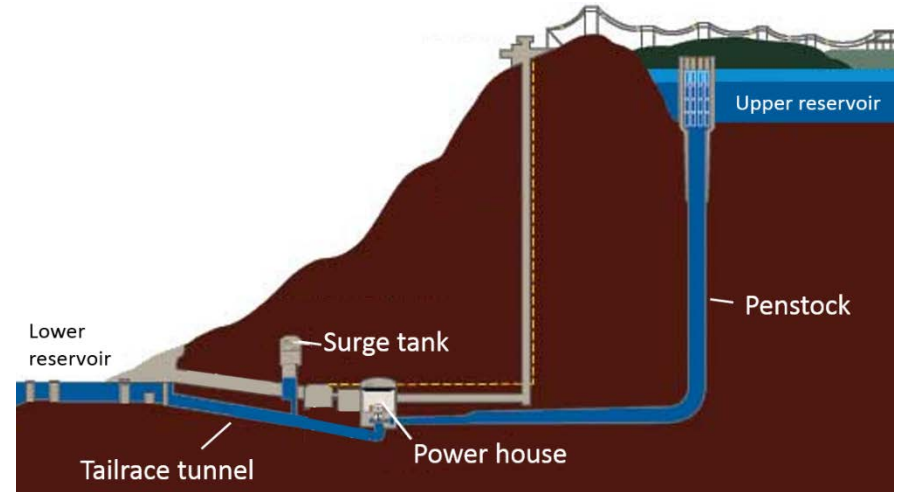
- Accumulator tanks to absorb high pressure transients during startup and mode changeover
- May be located on penstock or tailrace
- Especially important for longer tunnels
- *Hydraulic bypass capacitors*

PHES Components – Power House

28

□ **Power house**

- Contains pump/turbines and motor/generators
- Often underground
- Typically below the level of the lower reservoir to provide required pump inlet head
- Three possible configurations
 - **Binary set:** one pump/turbine and one motor/generator
 - **Ternary set:** one pump, one turbine, and one motor/generator
 - **Quaternary set:** separate pump, turbine, motor, and generator



29

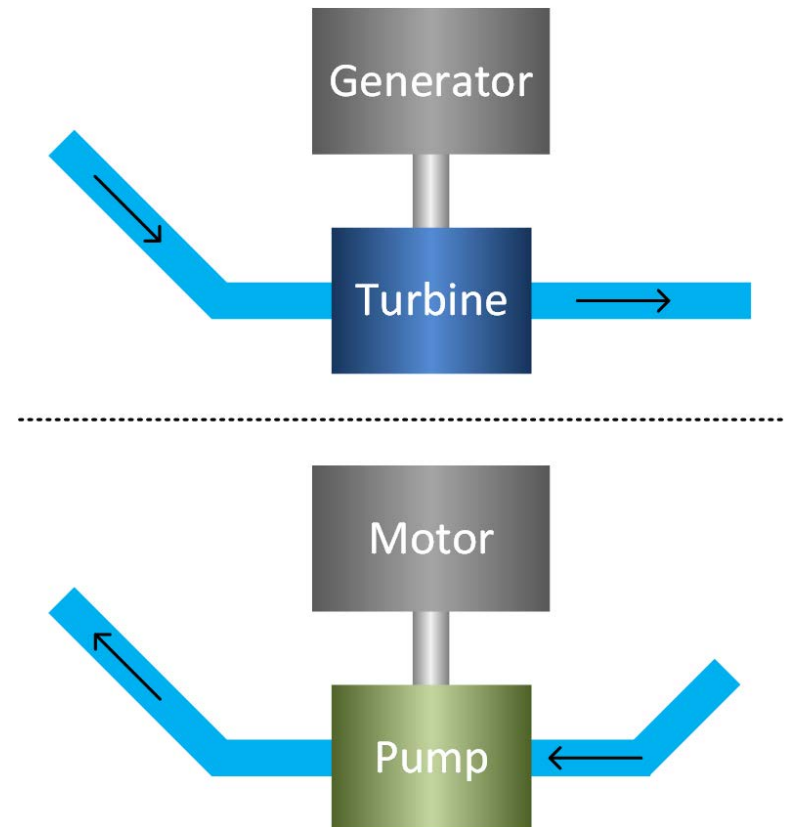
Power Plant Configurations

Power Plant Configurations – Quaternary Set

30

□ *Quaternary set*

- ▣ Pump driven by a motor
- ▣ Generator driven by a turbine
- ▣ Pump and turbine are completely decoupled
- ▣ Possibly separate penstocks/tailrace tunnels
- ▣ Most common configuration prior to 1920
- ▣ High equipment/infrastructure costs
- ▣ High efficiency
 - Pump and turbine designed to optimize individual performance

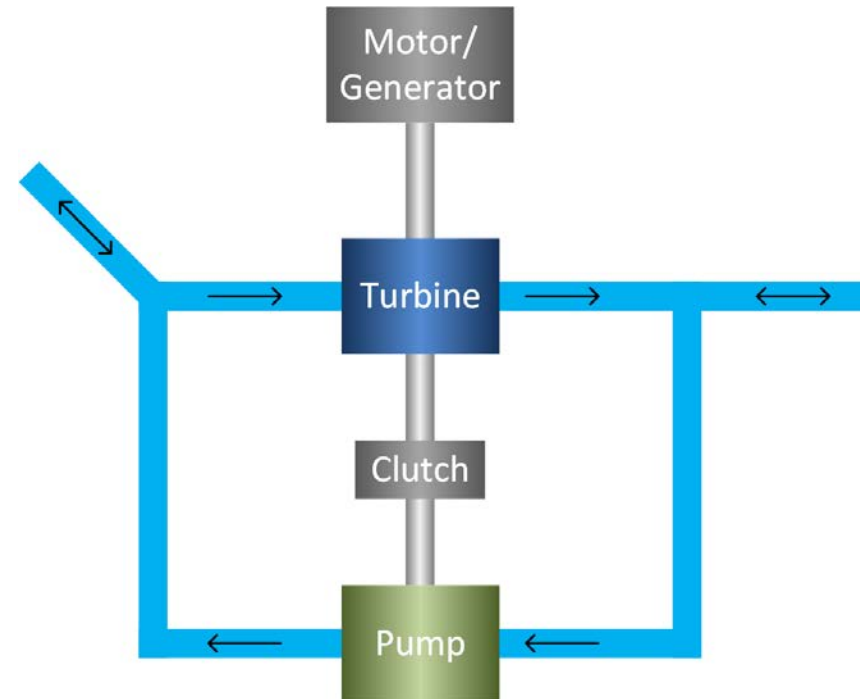


Power Plant Configurations – Ternary Set

31

□ ***Ternary set***

- Pump, turbine, and motor/generator all on a single shaft
 - Pump and turbine rotate in the same direction
- Turbine rigidly coupled to the motor/generator
- Pump coupled to shaft with a clutch
- Popular design 1920 – 1960s
- Nowadays, used when head exceeds the usable range of a single-stage pump/turbine
 - High-head turbines (e.g., Pelton) can be used
- Pump and turbine designs can be individually optimized

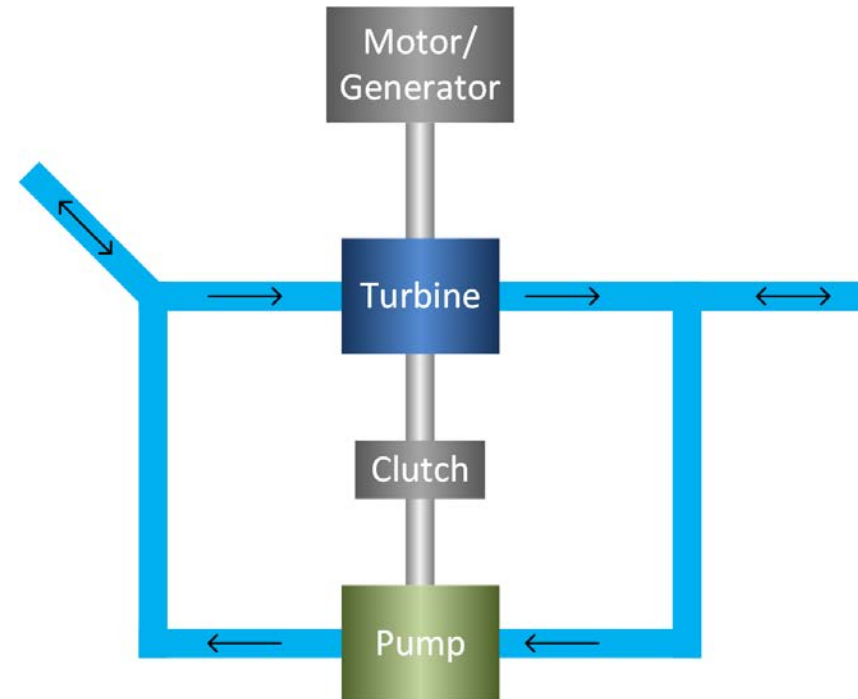


Power Plant Configurations – Ternary Set

32

□ ***Ternary set***

- Generating mode:
 - Turbine spins generator
 - Pump decoupled from the shaft and isolated with valves
- Pumping mode:
 - Motor turns the pump
 - Turbine spins in air, isolated with valves
- Both turbine and pump can operate simultaneously
- Turbine can be used for pump startup
 - Both spin in the same direction
 - Turbine brings pump up to speed and synchronized with grid, then shuts down
 - Changeover time reduced

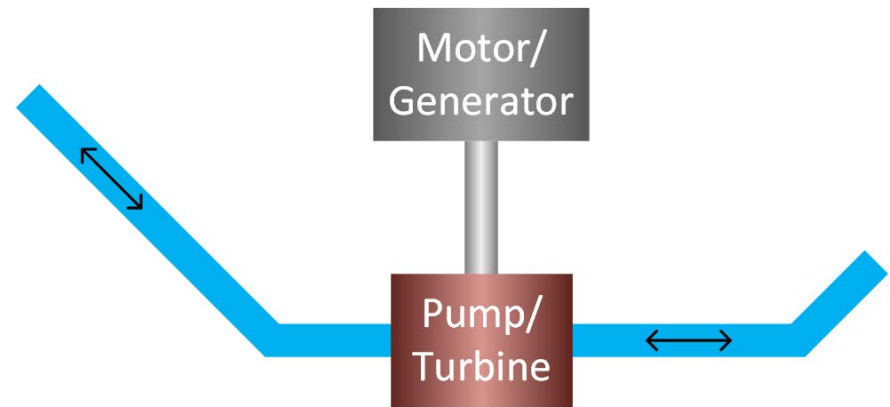


Power Plant Configurations – Binary Set

33

□ **Binary set**

- Single reversible pump/turbine coupled to a single motor/generator
- Most popular configuration for modern PHES
- Lowest cost configuration
 - Less equipment
 - Simplified hydraulic pathways
 - Fewer valves, gates, controls, etc.
- Lower efficiency than for ternary or quaternary sets
 - Pump/turbine runner design is a compromise between pump and turbine performance

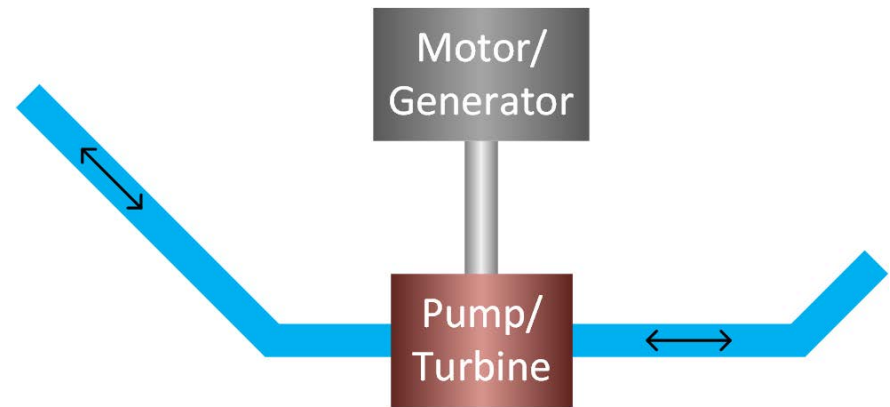


Power Plant Configurations – Binary Set

34

□ **Binary set**

- Rotation is in opposite directions for pumping and generating
- Shaft and motor/generator must change directions when changing modes
 - Slower changeover than for ternary or quaternary units
- Pump startup:
 - Pump/turbine runner dewatered and spinning in air
 - Motor brings pump up to speed and in synchronism with the grid before pumping of water begins



35

Turbines

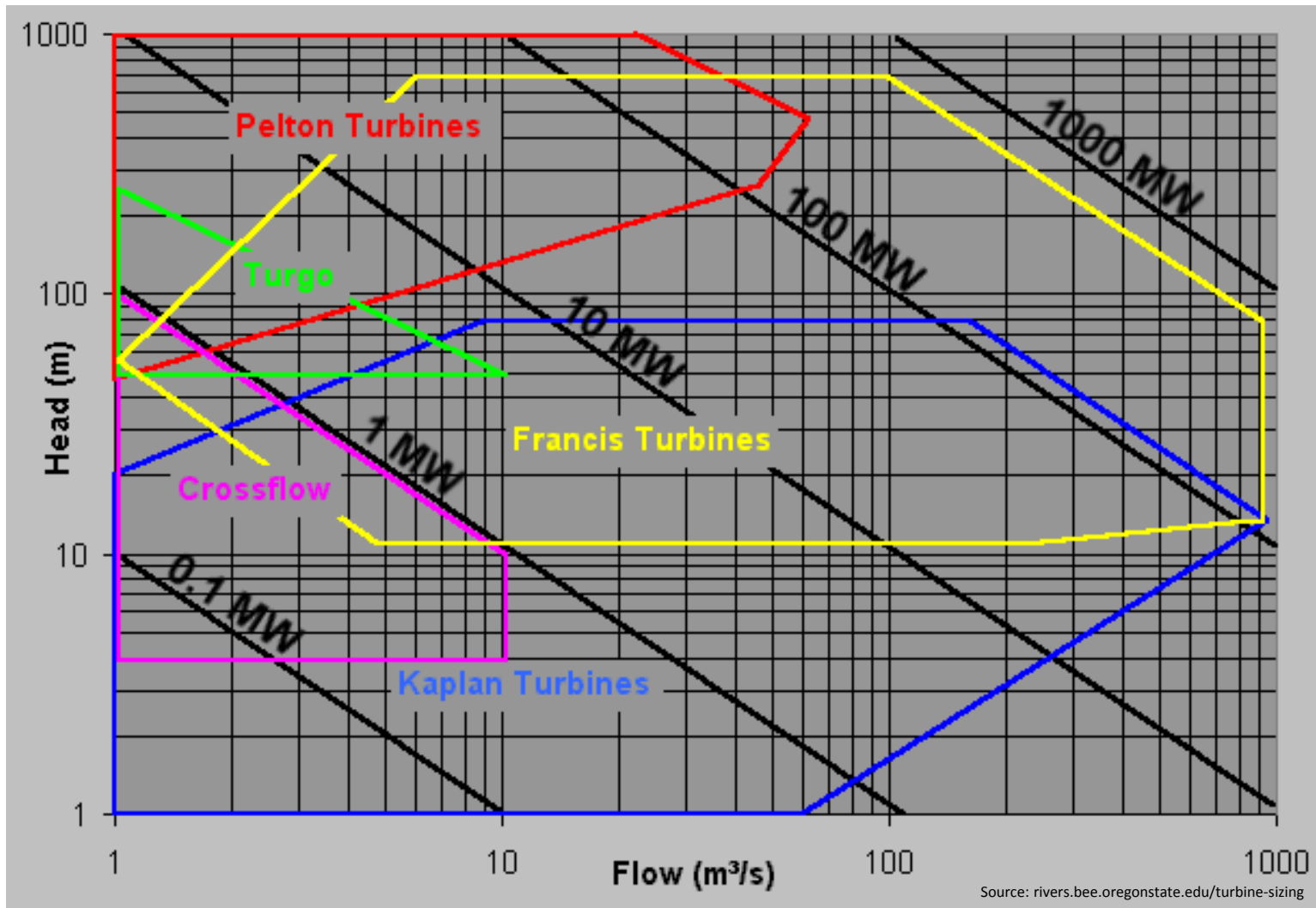
Turbines

36

- Hydro turbine design selection based on
 - ▣ Head
 - ▣ Flow rate
- PHES plants are typically sited to have large head
 - ▣ Energy density is proportional to head
 - ▣ Typically 100s of meters
- Reversible **Francis** pump/turbine
 - ▣ Most common turbine for PHES applications
 - ▣ Single-stage pump/turbines operate with heads up to 700 m
- For higher head:
 - ▣ Multi-stage pump/turbines
 - ▣ Ternary units with **Pelton** turbines

Turbine Selection

37



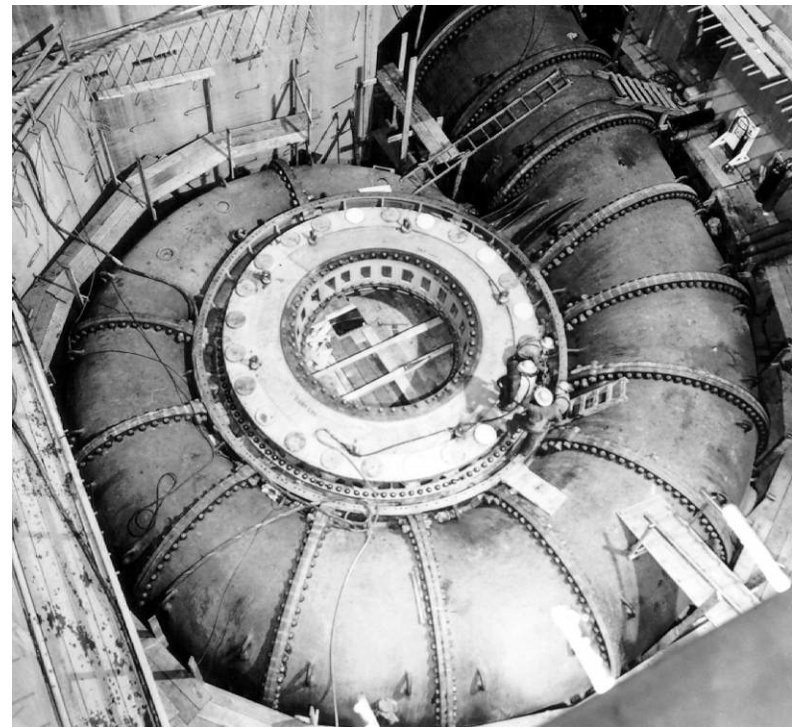
Francis Turbine – Components

38

□ ***Volute casing (scroll casing)***

- Spiral casing that feeds water from the penstock to the turbine runner
- Cross-sectional area decreases along the length of the casing
 - Constant flow rate maintained along the length

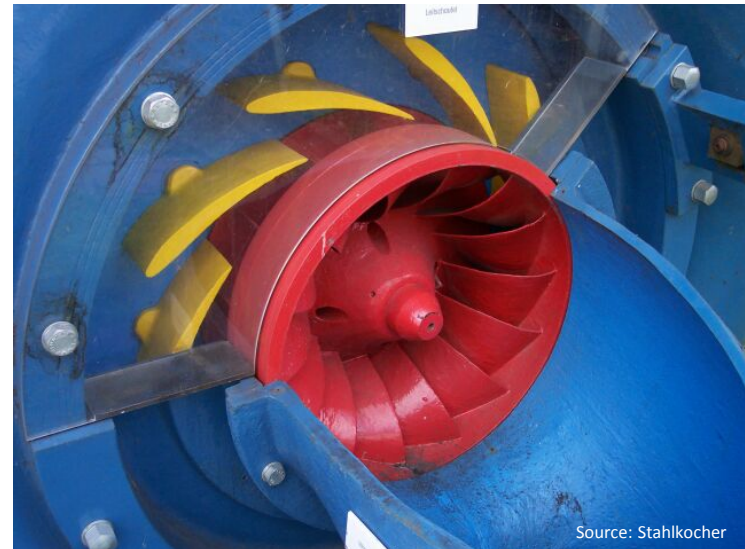
Francis turbine casing – Grand Coulee:



Francis Turbine – Components

39

- **Guide vanes** and **stay vanes**
 - ▣ Direct water flow from the casing into the runner
 - ▣ Stay vanes are fixed
 - ▣ Guide vanes, or **wicket gates**, are adjustable
 - Open and close to control flow rate
 - Power output modulated by controlling flow rate
 - Set fully open for pumping mode



Francis Turbine – Components

40

□ **Turbine runner**

□ **Reaction** turbine

- **Pressure energy** is extracted from the flow
- Pressure drops as flow passes through the runner
- Flow enters radially
- Flow exits axially
- Typically oriented with a vertical shaft



□ **Draft tube**

- Diffuser that guides exiting flow to the tailrace



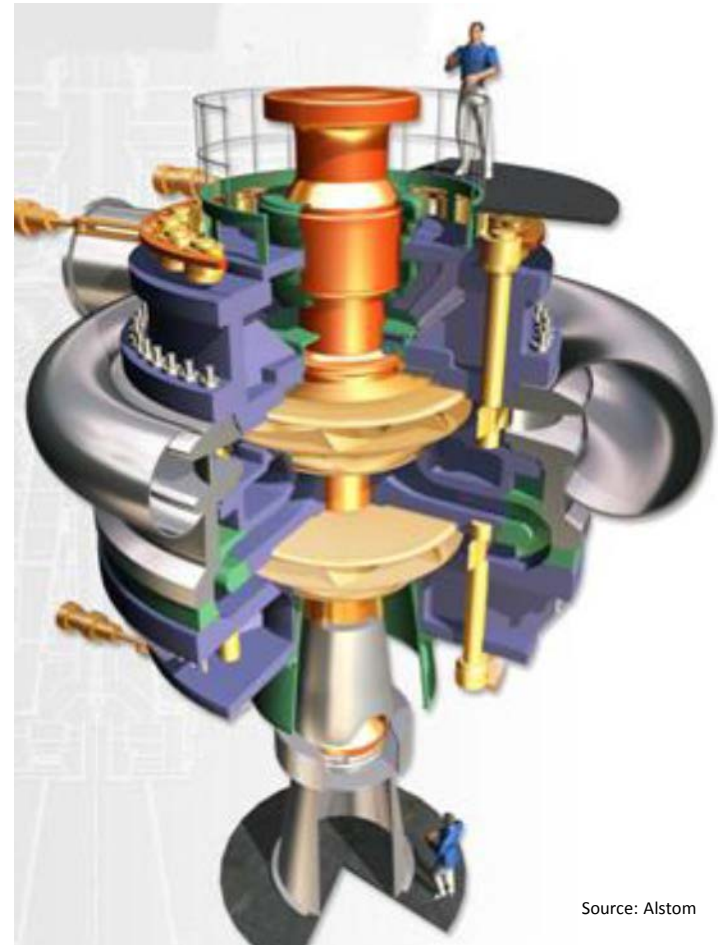
Source: Voith Siemens Hydro Power

High-Head PHES

41

- Options for heads in excess of 700 m:
 - ▣ ***Two-stage*** Francis pump/turbines
 - Typically no wicket gates in two-stage configuration
 - No mechanism for varying generating power
 - ▣ Ternary unit with ***Pelton turbine***

Two-stage pump/turbine:



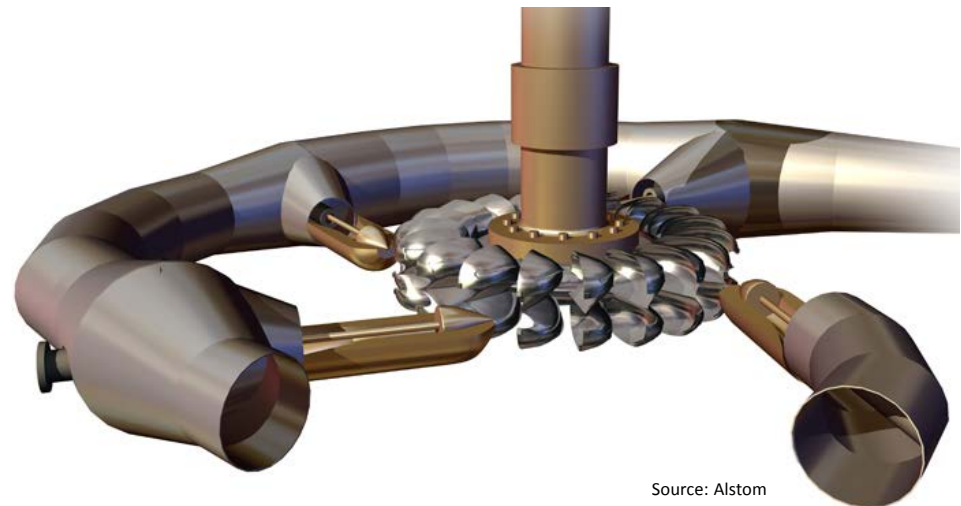
Source: Alstom

Pelton Turbines

42

□ ***Pelton Turbine***

- Suitable for heads up to 1000 m
- ***Impulse turbine***
 - Nozzles convert pressure energy to kinetic energy
 - High-velocity jets impinge on the runner at atmospheric pressure
 - Kinetic energy transferred to the runner
 - Water exits the turbine at low velocity
- Cannot be used for pumping
 - Used as part of a ternary set



43

Motor/Generator

Motor/Generator – Fixed-Speed

44

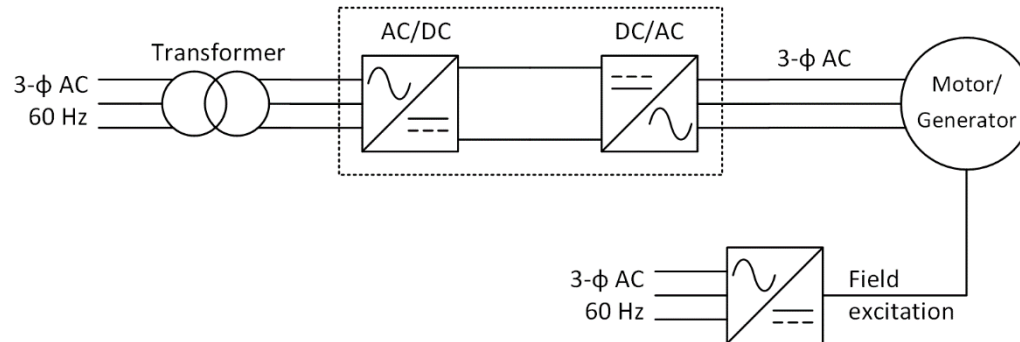
- Pump/turbine shaft connects to a motor/generator unit
 - Above the turbine runner in typical vertical configuration
 - Motor/generator type depends PHES category:
 - Fixed-speed pump/turbine
 - Variable-speed pump/turbine
-

- ***Fixed-speed pump/turbine***
 - Motor/generator operates at a fixed speed in both pumping and generating modes
 - ***Synchronous motor/generator***
 - Rotation is synchronous with the AC grid frequency
 - Stator windings connect to three-phase AC at grid frequency
 - Rotor windings fed with DC excitation current via slip rings
 - DC excitation current generated with thyristor AC/DC converters

Motor/Generator

45

- ***Variable-speed (adjustable-speed) pump/turbine***
 - ▣ Rotational speed of motor/generator is adjustable
 - ▣ Two options:
 - Variable speed using a synchronous motor/generator (singly-fed)
 - Doubly-fed asynchronous machine (DFAM)
- ***Variable-speed operation with synchronous motor/generator:***



- ▣ Motor driven with variable frequency
- ▣ Decoupled from grid frequency by back-to-back converters
- ▣ Converters must be rated for full motor/generator power
 - Large, expensive

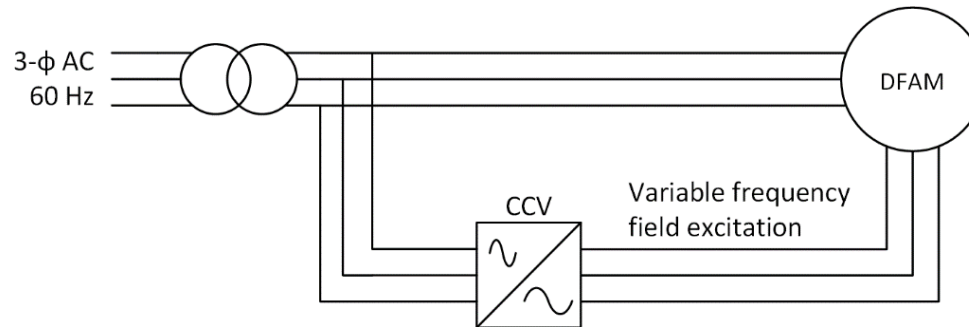
Motor/Generator – Variable-Speed

46

□ **Variable speed using doubly-fed asynchronous machines**

- Field excitation fed with variable, low-frequency AC, not DC as in synchronous machines
- Static frequency converter generates variable AC
 - Cycloconverter
 - Back-to-back voltage-source converters
- Typically small speed range (e.g., $\pm 10\%$)

□ **With *cycloconverter* generating variable-frequency excitation for rotor:**

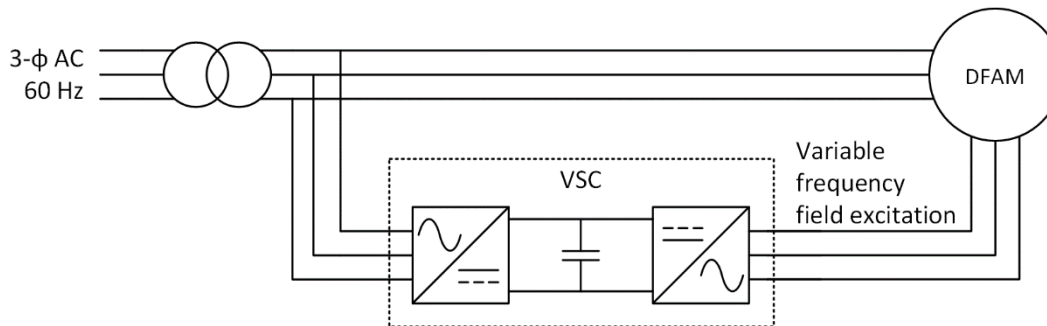


- Converters need not be sized for rated motor/generator power
 - Only supply lower-power excitation to the rotor

Motor/Generator – Variable-Speed

47

- DFAM with variable-frequency field excitation generated by back-to-back VSCs:



- The preferred configuration for large (>100 MW) PHES plants nowadays
-
- Advantages of variable-speed plants
 - Pump and turbine speeds can be independently varied to optimize efficiency over range of flow rate and head
 - Pumping power can be varied in addition to generating power

PHES for Frequency Regulation

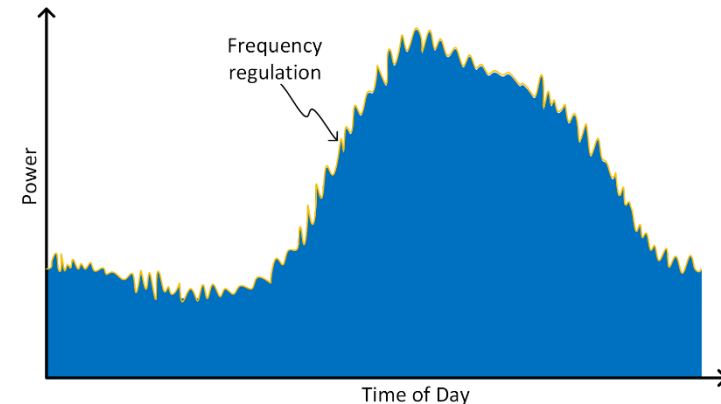
48

□ **Frequency regulation**

- Tracking short-term load variations to maintain grid frequency at 60 Hz (or 50 Hz)

□ PHES plants can provide frequency regulation

- Different for fixed- or variable-speed plants



□ **Fixed-speed plants**

■ Generating mode

- Frequency regulation provided by rapidly varying power output
- Power varied by using wicket gates to modulate flow rate
- Same as in conventional hydro plants

■ Pumping mode

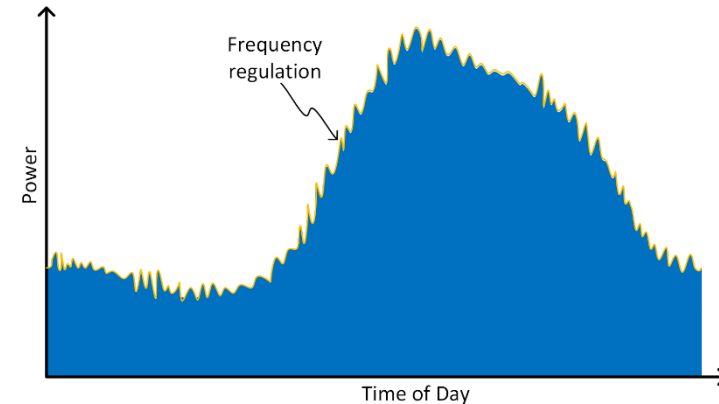
- Pump operates at rated power only – power input cannot be varied
- **No frequency regulation in pumping mode**

Frequency Regulation – Variable-Speed

49

□ ***Variable-speed plants***

- Pump speed can be varied over some range, e.g. $\pm 10\%$
- Pump power is proportional to pump speed ***cubed***
 - For $\pm 10\%$ speed variation, power is adjustable over $\pm 30\%$
- Power variation in pumping mode can track rapid load variations
- ***Frequency regulation can be provided in both modes of operation***



Frequency Regulation – Ternary Sets

50

□ ***Fixed-speed ternary sets***

■ Generating mode

- Wicket gates in turbine control flow rate to vary power output
- Pump disconnected from shaft

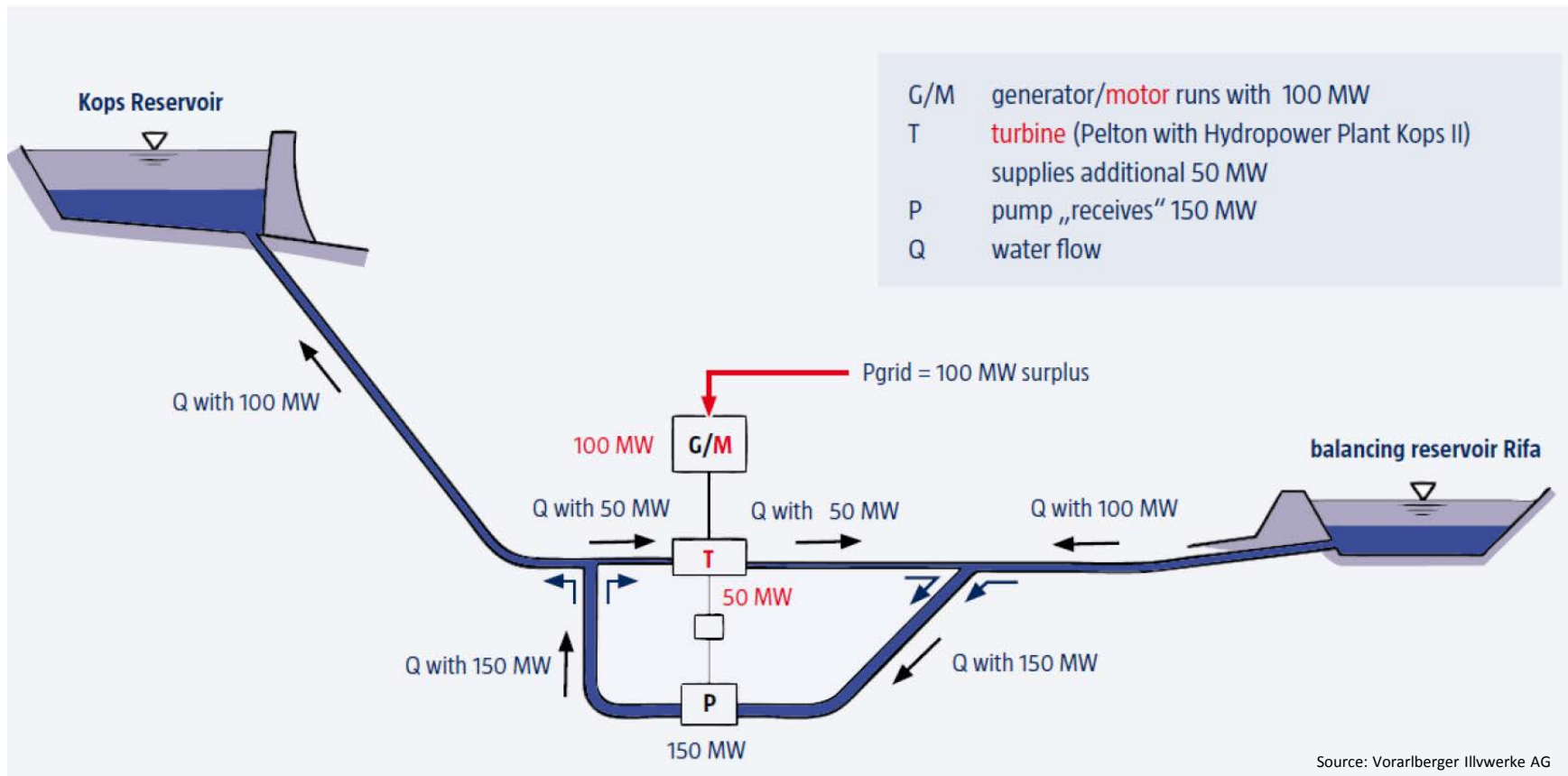
■ Pumping mode

- ***Hydraulic short circuit*** provides power modulation
- Pump and generator both turn on the shaft
- Pump operates at full load
- Generator operates at variable partial load

Hydraulic Short Circuit

51

□ Kops II PHES plant in Austrian Alps:



52

PHES Efficiency

PHES System Efficiency

53

□ Round-trip efficiency:

$$\eta_{rt} = \frac{E_{out}}{E_{in}} \cdot 100\%$$

where

- E_{in} is the electrical energy that flows in from the grid to the plant in pumping mode
- E_{out} is the electrical energy that flows from the plant to the grid in generating mode

■ Typical round-trip efficiency for PHES plants in the range of 70% – 80%

□ PHES loss mechanisms

- Transformer
- Motor/generator
- Pump/turbine
- Water conduit

PHES Losses

54

□ **Transformers**

- Pumped hydro plants connect to the AC electrical grid
 - Transformers step voltage between high voltage on the grid side to lower voltage at the motor/generator
- Transformer ***loss mechanisms***:
 - Winding resistance
 - Leakage flux
 - Hysteresis and eddy currents in the core
 - Magnetizing current – finite core permeability
- Power flows through transformers on the way into the storage plant and again on the way out
- Typical loss: $\sim 0.5\%$

PHES Losses

55

□ ***Motor/generator losses***

- ▣ Electrical resistance
- ▣ Mechanical friction
- ▣ Typical loss: $\sim 2\%$

□ ***Pump/turbine***

- ▣ Single runner in binary sets
 - Typically lower efficiency, particularly for fixed-speed operation – design of both compromised
- ▣ Separate runners in ternary, quaternary sets
 - Higher efficiency
- ▣ Typical loss: $\sim 7\% - 10\%$

PHES Losses

56

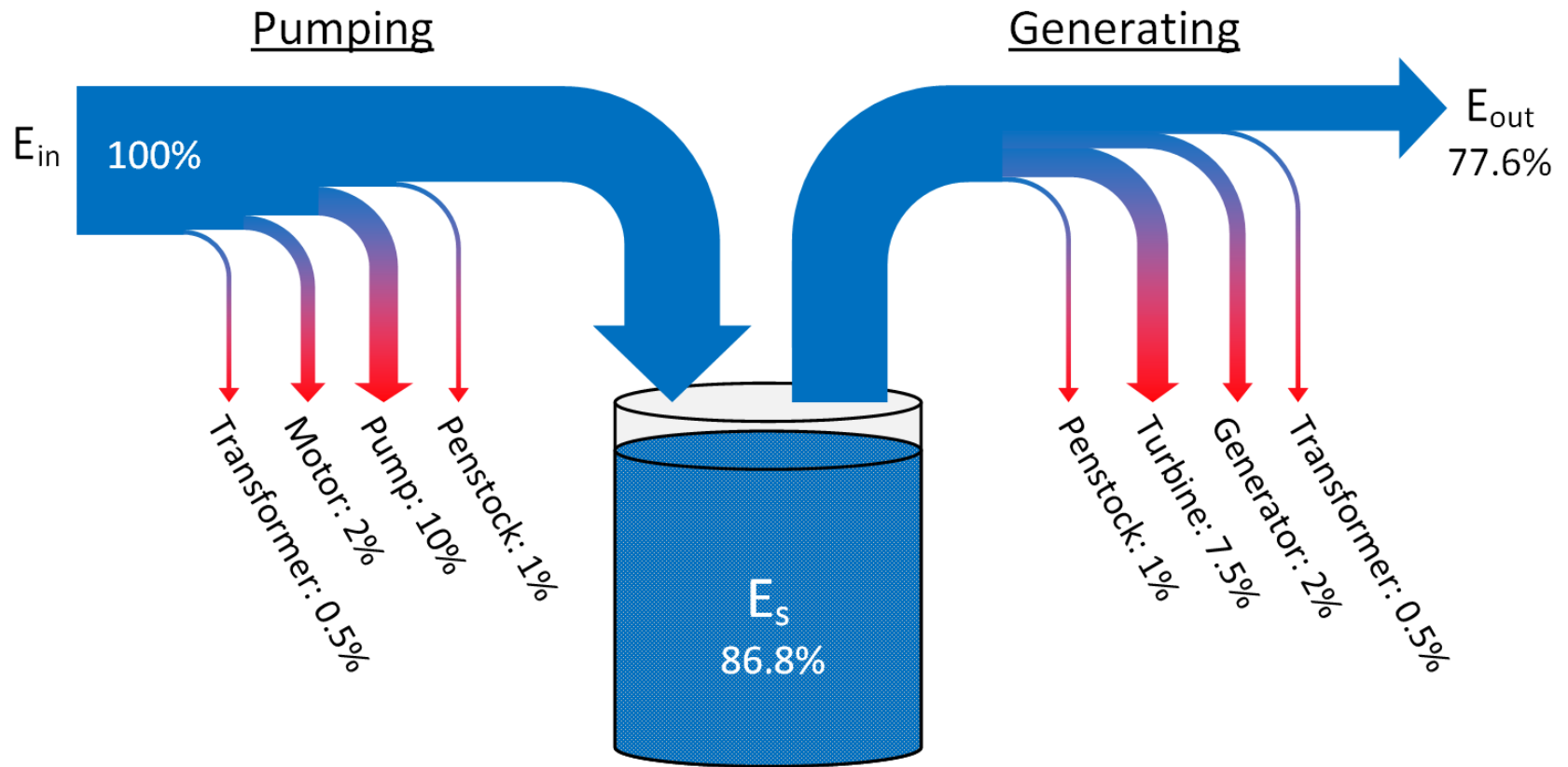
□ **Penstock**

- **Frictional loss** of water flowing through the conduit
 - Major losses along penstock
 - Minor losses from bends, penstock inlet, turbine inlet, etc.
- Dependent on
 - Flow velocity
 - Penstock diameter
 - Penstock length
 - Penstock lining – steel, concrete, etc.
- High head is desirable, but long penstocks are not
 - Steeper penstocks reduce frictional losses for a given head
 - Typical length-to-head ratio: 4:1 – 12:1
- Typical loss: ~1%

PHES Losses

57

□ Typical losses for PHES:



Pumping-Mode Efficiency

58

- ***Efficiency of the pumping operation*** is given by

$$\eta_p = \frac{E_s}{E_{in}} \cdot 100\%$$

where

- E_s is the energy stored
 - Potential energy of the volume of water, V_u , pumped to the upper reservoir

$$E_s = V_u \rho g h$$

- E_{in} is the energy input from the grid during the pumping operation
- The mechanical energy input to the pump is

$$E_{in,pump} = E_{in} \cdot \eta_{trans} \cdot \eta_{motor}$$

where

- η_{trans} and η_{motor} are the efficiencies of the transformer and motor, respectively

Pumping-Mode Efficiency

59

- The volume of water pumped to the upper reservoir is

$$V_u = \frac{E_{in,pump}}{\rho gh} \cdot \eta_{pump} \cdot \eta_{pipe,p}$$

where

- η_{pump} is the pump efficiency
- $\eta_{pipe,p}$ is the penstock efficiency in pumping mode
- So, the total pumped volume of water is

$$V_u = \frac{E_{in}}{\rho gh} \cdot \eta_{trans} \cdot \eta_{motor} \cdot \eta_{pump} \cdot \eta_{pipe,p}$$

- The **pumping-mode efficiency** is therefore:

$$\eta_p = \frac{E_s}{E_{in}} = \eta_{trans} \cdot \eta_{motor} \cdot \eta_{pump} \cdot \eta_{pipe,p}$$

Generating-Mode Efficiency

60

- Efficiency of the generating operation is given by

$$\eta_g = \frac{E_{out}}{E_s} \cdot 100\%$$

- Due to frictional losses in the penstock, the hydraulic energy that reaches the turbine is

$$E_{in,t} = E_s \cdot \eta_{pipe,g}$$

- The amount of rotational energy at the turbine output/generator input is

$$E_{in,g} = E_{in,t} \cdot \eta_t = E_s \cdot \eta_{pipe,g} \cdot \eta_t$$

- After generator and step-up transformer losses, the energy output to the grid is

$$E_{out} = E_{in,g} \cdot \eta_{gen} \cdot \eta_{trans}$$

$$E_{out} = E_s \cdot \eta_{pipe,g} \cdot \eta_t \cdot \eta_{gen} \cdot \eta_{trans}$$

Generating-Mode Efficiency

61

- **Generating mode efficiency** is

$$\eta_g = \frac{E_{out}}{E_s} = \eta_{pipe,g} \cdot \eta_t \cdot \eta_{gen} \cdot \eta_{trans}$$

- The **overall round-trip efficiency** is therefore

$$\eta_{rt} = \frac{E_{out}}{E_{in}} = \eta_p \cdot \eta_g$$

$$\eta_{rt} = \eta_{trans} \cdot \eta_{motor} \cdot \eta_{pump} \cdot \eta_{pipe,p} \cdot \eta_{pipe,g} \cdot \eta_t \cdot \eta_{gen} \cdot \eta_{trans}$$

Pumping and Generating Times

62

- Due to losses, charging/discharging times differ, even for equal grid-side power input/output
 - ▣ Energy flows in from the grid faster than it is stored in the upper reservoir
 - ▣ Energy flows out of storage faster than it is delivered to the grid

- **Charging (pumping) time:**

$$t_p = \frac{E_{in}}{P_{in}} = \frac{E_s}{\eta_p P_{in}}$$

$$t_p = \frac{V_u \rho g h}{\eta_p P_{in}}$$

- **Discharging (generating) time:**

$$t_g = \frac{E_{out}}{P_{out}} = \frac{E_s \eta_g}{P_{out}}$$

$$t_g = \frac{V_u \rho g h \eta_g}{P_{out}}$$

Pumping and Generating Times

63

- Ratio of generation to pumping time:

$$\frac{t_g}{t_p} = \frac{V_u \rho g h \eta_g}{P_{out}} \frac{\eta_p P_{in}}{V_u \rho g h} = \frac{P_{in}}{P_{out}} \eta_g \eta_p$$

$$\boxed{\frac{t_g}{t_p} = \frac{P_{in}}{P_{out}} \eta_{rt}}$$

- For equal input and output power, this becomes

$$\boxed{\frac{t_g}{t_p} = \eta_{rt}}$$

- That is, the ratio of discharging to charging time is equal to the round-trip efficiency

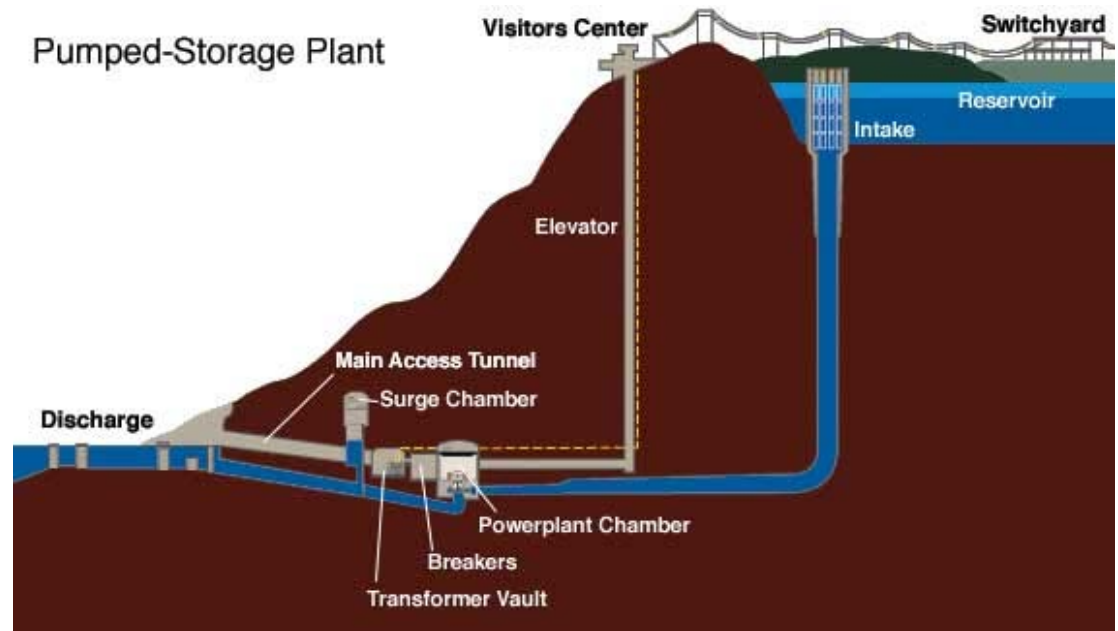
Example PHES Projects

Raccoon Mountain

65

- ❑ Marion County, TN
- ❑ Open-loop PHES
 - Mountaintop upper reservoir
 - $46 \times 10^6 \text{ m}^3$ of water
 - Tennessee River is lower reservoir
- ❑ **Power:** 1652 MW
 - 4 x 413 MW pump/turbine units
- ❑ **Energy:** 36.3 GWh
- ❑ **Pump/turbines:** single-stage reversible Francis
- ❑ **RT efficiency:** 79%
- ❑ Commissioned: 1978
- ❑ Penstock diameter: 10.7 m
- ❑ **Head:** 273 – 317 m

- ❑ **Generating time:** 22 hours
- ❑ **Pumping time:** 28 hours
- ❑ Usage: peaking generation, grid balancing



Bath County

66

- ❑ Open-loop PHES
- ❑ **World's largest** PHES facility
- ❑ Bath County, VA
 - Upper reservoir: $44 \times 10^6 \text{ m}^3$
 - Lower reservoir: $34 \times 10^6 \text{ m}^3$
- ❑ **Generating power:** 3 GW
 - 6 x 500 MW
- ❑ **Pumping power:** 2.88 GW
 - 6 x 480 MW
- ❑ **Energy:** 30.9 GWh
- ❑ **Generating time:** 10.3 hrs
- ❑ **RT efficiency:** 78%
- ❑ **Head:** 350 – 400 m
- ❑ Commissioned: 1985
- ❑ **Pump/turbines:** single-stage reversible Francis
- ❑ Penstocks:
 - 3 x 8.7 m x 1000 m tunnels to
 - 3 x 8.7 m 300 m vertical shafts to
 - 6 x 5.5 m x 300 m tunnels
- ❑ **Generating flow rate:** $850 \text{ m}^3/\text{s}$
- ❑ **Pumping flow rate:** $800 \text{ m}^3/\text{s}$
- ❑ Usage: daily load following and peaking
 - Pumping at night, generating during the day



Goldisthal

67

- Open-loop PHES
- Goldisthal, Germany
 - ▣ Upper reservoir: $12 \times 10^6 \text{ m}^3$
 - ▣ Lower reservoir: $18.3 \times 10^6 \text{ m}^3$
- **Power:** 1060 MW
 - ▣ 4 x 265 MW
- **Energy:** 8.48 GWh
- **Generating time:** 8 hrs
- **RT efficiency:** >80%
- **Head:** 280 – 325 m
- Commissioned: 2004
- **Pump/turbines:**
 - ▣ single-stage reversible Francis
 - ▣ Two fixed-speed, two **adjustable-speed**
- Penstocks: 2 x 6.2 m x 820 m tunnels
- Tailrace tunnels: 2 x 8.2 m x 277 m
- **Max flow rate:**
 - ▣ Generating: $400 \text{ m}^3/\text{s}$
 - ▣ Pumping: $320 \text{ m}^3/\text{s}$
- Usage: load-following, peak generation, regulation, black start



68

Rail Energy Storage

Disadvantages of PHES

69

□ ***Disadvantages of PHES***

▣ Environmental issues

- Water usage
- River/habitat disruption

▣ Head variation

- Pressure drops as upper reservoir drains
- Efficiency may vary throughout charge/discharge cycle
- Particularly an issue for lower-head plants with steep, narrow upper reservoirs

▣ Siting options are limited

- Available water
- Favorable topography
- Large land area

□ Possible alternative potential energy storage:

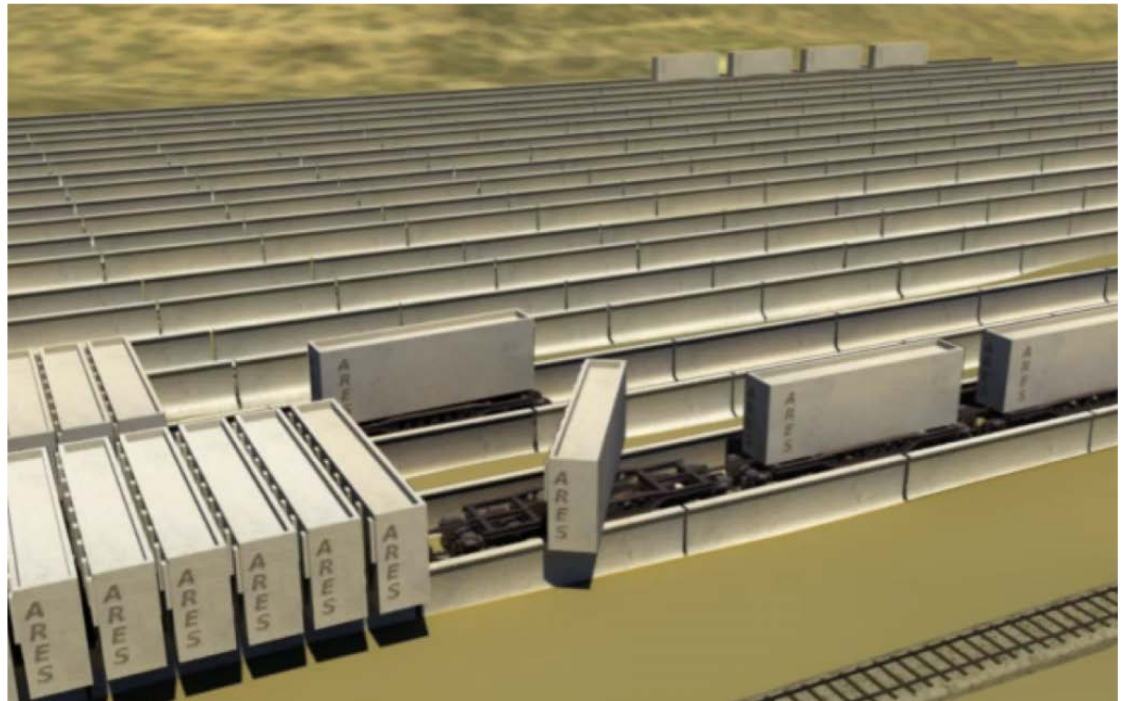
▣ ***Rail energy storage***

Rail Energy Storage

70

□ ***Rail energy storage***

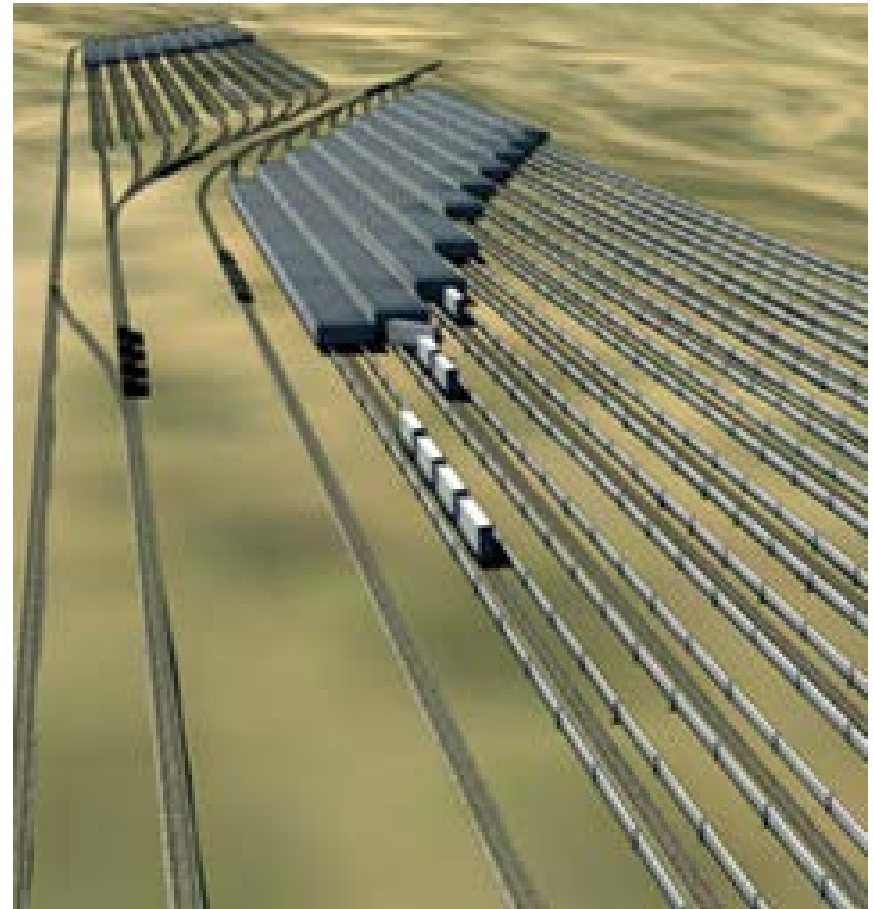
- Electric-motor-driven railcars
- Weights are shuttled up and down an incline between upper and lower storage yards
- Power input drives motors to move weights up the track
- Regenerative braking on the way down supplies power to the grid
- Weights are loaded and unloaded at storage yards
 - Large quantities of energy can be stored with few trains



Advantages of Rail Energy Storage

71

- More siting options than for PHES
 - ▣ Open space
 - ▣ Elevation change
 - ▣ No need for water or topography conducive to reservoirs
- Lower capital cost than PHES
- Easily scalable
- Efficient
 - ▣ RT efficiency: 78% - 86%
 - ▣ Constant efficiency, independent of SoC
- No standby losses
 - ▣ No evaporation/leakage



Rail Energy Storage

72

□ ARES North America

- ▣ Scale prototype project constructed in Tehachapi, CA
- ▣ 50 MW frequency regulation project planned for southern Nevada

□ ***ARES Nevada***

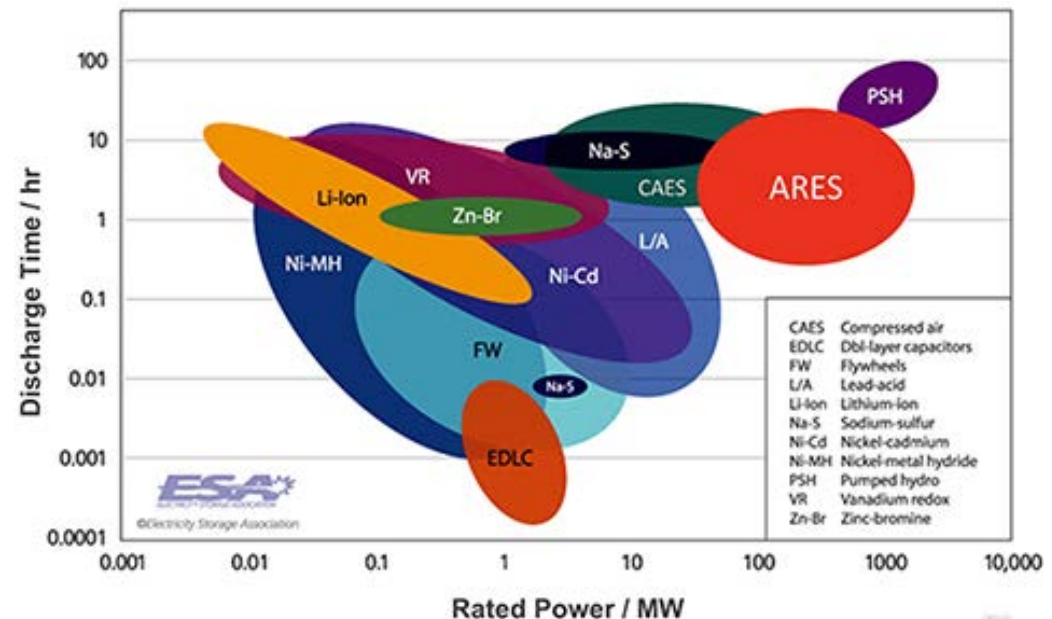
- ▣ Location: BLM land, Pahrump, NV
- ▣ Power: 50 MW
- ▣ Energy: 12.5 MWh
- ▣ Generating time at rated power: 15 min
- ▣ Track length: 9 km (5.5 mi)
- ▣ Elevation difference: 610 m (2000 ft)
- ▣ Total mass: 8.7×10^6 kg (9600 US tons)
- ▣ Footprint: 46 acres
- ▣ Status: licensing, permitting, and environmental review phase

Rail Energy Storage

73

□ Three categories of rail energy storage plants proposed by ARES:

- Small
 - 20 – 50 MW
 - Ancillary services only
- Intermediate
 - 50 – 200 MW
 - Ancillary services, integration of renewables
- Grid-scale
 - 200 MW – 3 GW
 - 4 – 16 hours of storage at full power



Rail Energy Storage

74

- Conceptual grid-scale storage facility (as proposed by ARES)
 - Power: 670 MW
 - Energy: 5360 MWh
 - Discharge time: 8 hr
 - Elevation differential: 915 m (3000 ft)
 - Five tracks
 - Length: 13 km (8 mi)
 - Grade: 7.5%
 - 140 4-car shuttle trains
 - 11,400 concrete weights
 - Mass of each: 212×10^3 kg (234 US tons)
 - Total mass: 2.42×10^9 kg (2.67×10^6 tons)
 - Capital costs:
 - \$1350/kW
 - \$168/kWh